ROBUST CONTROLLER DESIGN IN THE FREQUENCY DOMAIN

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Abstract: In this paper a novel design technique is proposed to guarantee a required performance of the full system by applying the independent design to equivalent subsystems.

Keywords: Linear system, Robust control, Frequency domain

1 INTRODUCTION

During the last decades robustness has been recognized as a key issue in the analysis and design of control systems. The history of robust control design based on small-gain-like robustness condition started developing with the pioneering work of Zames where robust control design problem has been formulated as an optimization problem. Only at the end of the 1980's was found a practical solution to this problem. It is worth to mention some algebraic approaches which followed the seminal works of [KHARITONOV, 1979], [BHATTACHARYYA, 1995] and [BARLET, 1988].

In this paper we focus our attention on two robust design problem. First the problem of robust stabilization of an uncertain single input-single output (SISO) plant described by the transfer function with linear or multilinear interval systems is considered. Multiple-input-multiple output (MIMO) systems usually arise as an interconnection of a £nite number of subsystems. In case of such systems practical reasons often make restrictions on controller structure necessary or reasonable. The controller split into several local feedbacks becomes a decentralized controller. With the come up of robust frequency domain approach in the 80's several practice oriented techniques were developed, see [SKOGESTAD,1996], [KOZAKOVA, 2003]. The decentralized controller design comprises two steps: 1. selection of control con£guration, 2. design of local controllers. In the second part of this paper we deal with the Step 2. The independent design approach has been adopted. In the independent design used in sequel local controllers are designed without considering interactions with other subsystems. In this paper a novel design technique is proposed to guarantee a required performance of the full system by applying the independent design to the equivalent subsystems.

2 PRELIMINARIES AND MODEL UNCERTAINTIES

Consider a closed-loop system comprising the transfer function matrix of the plant $G(s) \in R^{m \times m}$ and the controller $R(s) \in R^{m \times m}$ in the standard feedback configuration, Fig.1

where w, u, y, e are respectively vectors of reference, control input, output and control error of compatible dimensions.

The problem addressed in this paper is the design of a robust decentralized controller

$$R(s) = diag\{R_{ii}(s)\}_{m \times m} \tag{1}$$

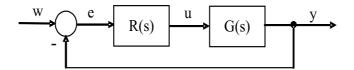


Figure 1 – Standard feedback con£guration

that guarantees closed-loop stability and performance over the entire operating range of the controlled plant G(s).

Let the plant be given by a set of N transfer function matrices identi£ed in different working points

$$G^{k}(s) = \{G_{ij}^{k}(s)\}_{m \times m}, \quad k = 1, 2, ...N$$

with

$$G_{ij}^{k}(s) = \frac{y_i^{k}(s)}{u_i^{k}(s)}$$
 $i, j = 1, 2, ..., m$

where $y_i^k(s)$ is the i-th output and $u_i^k(s)$ is the j-th plant input in the k th experiment.

Uncertainty associated with a real system model can be described in various ways. There are following types of uncertainty models encountered in the literature:

structured uncertainty (parametric uncertainty: interval model, af£ne model, multilinear and nonlinear; dynamic uncertainty with known structure), and

unstructured uncertainty (additive, multiplicative and inverse uncertainty models which include parametric and dynamic uncertainty with unknown structure).

In practice, at a particular frequency the transfer function magnitude and phase are supposed to lie within a disc-shaped region around the nominal transfer function $G_N(s)$. Over a given frequency range, these disc-shaped regions can be generated by the following forms: additive (2), multiplicative input (3) and multiplicative output (4) uncertainties, as well as by the inverse forms [SKOGESTAD, 1996]. In the sequel just the £rst three uncertainty forms will be considered, respectively to describe the uncertain plant G(s).

$$\Pi_a: G(s) = G_N(s) + l_a(s)\Delta(s) \quad l_a(s) = \max_k \sigma_M[G^k(s) - G_N(s)]$$
(2)

$$\Pi_i: G(s) = G_N(s)[I + l_i(s)\Delta(s)] \quad l_i(s) = \max_k \sigma_M\{G_N(s)^{-1}[G^k(s) - G_N(s)]\}$$
(3)

$$\Pi_o: \quad G(s) = [I + l_o(s)\Delta(s)]G_N(s) \quad l_o(s) = \max_k \sigma_M\{[G^k(s) - G_N(s)]G_N(s)^{-1}\}$$
 (4)

where $\sigma_M(.)$ is the maximum singular value of the corresponding matrix; $\Delta(s)$ is the uncertainty matrix that satis£es

$$\Delta(s)^T \Delta(s) \le I \tag{5}$$

For the SISO case, m=1, the multiplicative input and output uncertainties equal. In the frequency domain, uncertain SISO systems can be described using either of the above uncertainty types as well as the following ones:

-linear interval, (linear) af£ne, multilinear and nonlinear uncertainties.

Consider a SISO plant (m = 1) and a controller with transfer functions in the following forms

$$G(s) = \frac{P_1(s)}{P_2(s)} \quad R(s) = \frac{R_1(s)}{R_2(s)} \tag{6}$$

where $P_i(s)$, i = 1, 2 are a linear interval polynomials

$$P_i(s) = p_{oi} + p_{1i}s + \dots + p_{n:i}s^{n_i}$$
(7)

with

$$p_{ji} \in \langle p_{ji}, \overline{p_{ji}} \rangle$$
 $i = 1, 2; j = 1, 2, ..., n_i$

Let us de£ne the corresponding parameter uncertainty box

$$Q_i = \{ p_i : p_{ji} \le p_{ji} \le \overline{p_{ji}}, i = 1, 2; j = 0, 1, 2, ..., n_i \}$$
(8)

The global parameter uncertainty box is then

$$Q = Q_1 \times Q_2$$

The following assumptions about the linear interval polynomials are considered:

- Elements of $p_i \in Q_i$, i = 1, 2 are perturbed independently of each other. Equivalently, Q is $(n_1 + n_2)$ axis parallel rectangular box.
- Characteristic polynomials of the plant and the controller are of the same degree.

According to [BHATTACHARYYA, 1995] the closed-loop stability problem can be solved using the **Generalized Kharitonov Theorem**.

Theorem 1

For a given $R(s) = [R_1(s)R_2(s)]$ of real polymials:

R(s) stabilizes the linear interval polynomials $P(s) = [P_1(s)P_2(s)]$ for all $p \in Q$ if and only if the controller stabilizes the extremal transfer function

$$G_E(s) = \{ \frac{K_1(s)}{S_2(s)} \bigcup \frac{S_1(s)}{K_2(s)} \}$$
(9)

Moreover, if the controller polynomials $R_i(s)$, i = 1, 2 are of the form

$$R_i(s) = s^{t_i}(a_i s + b_i) U_i(s) Z_i(s)$$

$$\tag{10}$$

then it is sufficient if the controller R(s) stabilizes the Kharitonov transfer function

$$G_K(s) = \frac{K_1(s)}{K_2(s)} \tag{11}$$

where $K_i(s) = \{K_i(s)^1, K_i(s)^2, K_i(s)^3, K_i(s)^4\}$ stand for Kharitonov polynomials corresponding to each $P_i(s)$ and

$$S_i(s) = \{ [K_i(s)^1, K_i(s)^2], [K_i(s)^1, K_i(s)^3], [K_i(s)^2, K_i(s)^4], [K_i(s)^3, K_i(s)^4] \}$$
 (12)

stand for *Kharitonov segments* for corresponding $P_i(s)$;

 $U_i(s)$ is anti-Hurwitz polynomial;

 $Z_i(s)$ is an even or odd polynomial;

 a_i, b_i are positive numbers and $t_i \ge 0$ is an integer.

Note that

$$S_i(s)^1 = \lambda K_i(s)^1 + (1 - \lambda)K_i(s)^2 \quad \lambda \in \{0 \quad 1 > .$$

Let the plant transfer function of G(s) be written in the following af£ne form

$$G(s) = \frac{P_1(s)}{P_2(s)} = \frac{P_{0,1}(s) + \sum_{i=1}^{p} P_{i,1}(s)q_i}{P_{0,2}(s) + \sum_{i=1}^{p} P_{i,2}(s)q_i}$$
(13)

where $P_{j,1}(s)$, $P_{j,2}(s)$, j = 0, 1, ..., p are real polynomials with constant parameters and the uncertainty parameter q_i is from the interval

$$q_i \in \langle q_i, \overline{q_i} \rangle$$
 $i = 1, 2, ..., p$

The description (13) represents a polytope of linear systems with the vertices

$$G_{vj}(s) = \frac{P_{v1,j}(s)}{P_{v2,j}(s)} \quad j = 1, 2, ..., N; N = 2^p$$
(14)

computed for different variables $q_i(s)$, i=1,2,...,p taking alternatively their maximum $\overline{q_i}$ and minimum values q_i . Based on the Edge theorem [BARLETT,1988] the following results can be obtained

Theorem 2

The controller $R(s) = [R_1(s)R_2(s)]$ with real polynomials stabilizes the af£ne system (13) for all $q \in Q$ if and only if the controller stabilizes the following extremal transfer function

$$G_P(s) = \frac{\lambda P_{v1,i} + (1-\lambda)P_{v1,j}}{\lambda P_{v2,i} + (1-\lambda)P_{v2,j}} \quad \lambda \in <0, 1 > i \neq j, i, j = 1, 2, ..., p2^{p-1}$$
(15)

Both i and j have to be taken as e vertices numbers of corresponding edges. In general, the sets of extremal transfer functions (9) and (15) are quite different. Whilst the number of $G_E(s)$ is equal to 32,

the number of $G_P(s)$ depends exponentially on the number of uncertain parameters q_i . For the case of multilinear uncertainty consider the following uncertain plant transfer function

$$G(s) = \frac{P_{11}(s)P_{12}(s)...P_{1n}(s)}{P_{21}(s)P_{22}(s)...P_{2d}(s)}$$
(16)

where $P_{ij}(s)$, i = 1, 2; j = 1, 2, ..., n(d) belong to a linear interval polynomial specified as follows

$$p_k^{i,j} \in < p_k^{i,j}, \overline{p_k^{i,j}} >, i = 1, 2; j = 1, 2, ..., n(d), k = 0, 1, ..., n_{ij}(d_{ij})$$

with independently varying parameters.

Let $K_{ij}(s)$ and $S_{ij}(s)$ denote Kharitonov polynomials [KHARITONOV,1979] and Kharitonov segments of corresponding $P_{ij}(s)$, respectively. The following theorem holds [BHATTACHARYYA, 1995].

Theorem 3

The controller $R(s) = [R_1(s)R_2(s)]$ stabilizes the multilinear system (16) for the uncertainty box if and only if the polynomials R(s) stabilizes the following extremal transfer function

$$M_E(s) = \left\{ \frac{S_{11}(s)...S_{1n}(s)}{K_{21}(s)...K_{2d}(s)} \bigcup \frac{K_{11}(s)...K_{1n}(s)}{S_{21}(s)...S_{2d}(s)} \right\}$$
(17)

For the sake of limited space, other uncertainty types will not be considered here. For more detail see [BHATTAcharyya, 1995],[GRMAN, 2005].

3 ROBUST CONTROLLER DESIGN

This section deals with the robust controller design for both MIMO and SISO systems using either of the uncertainty types (2) or (3) or (4). Standard feedback con£guration for the uncertain system with additive uncertainty is depicted in Fig. 2.

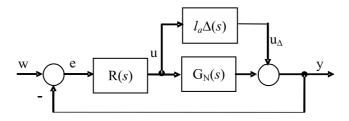


Figure 2 – Standard feedback con£guration with additive uncertainty

The above block diagram can easily be transformed in to the $M-\Delta$ structure in Fig.3 [SKOGESTAD, 1996].

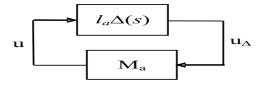


Figure 3 – M – Δ structure of closed-loop system

For different uncertainty types the following results have been obtained

$$M_{a}(s) = -[I + R(s)G_{N}(s)]^{-1}R(s)$$

$$M_{i}(s) = -[I + R(s)G_{N}(s)]^{-1}R(s)G_{N}(s) \quad \forall s \in D$$

$$M_{o}(s) = -G_{N}(s)R(s)[I + G_{N}(s)R(s)]^{-1}$$
(18)

Robust stability conditions are given in following theorem [SKOGESTAD, 1996].

Theorem 4

Assume that the nominal closed-loop system $M_k(s)$, k = a, i, o is stable and the uncertainties satisfy the following inequality

$$0 < l_k(s) \le l_{km}(s) \quad k = a, i, o.$$
 (19)

Then the $M-\Delta$ system is stable for all uncertainty models $l_k(s), k=a,i,o$ satisfying (19) if and only if

$$\sigma_M(M_k(s)) < \frac{1}{l_k(s)} \tag{20}$$

The resulting robust decentralized controller design procedure consist of two steps:

- designing a controller R(s) which guarantees stability and performance for nominal plant $G_N(s)$ (nominal stability)
- veri£cation of the robust stability condition (20)

Nominal closed-loop stability under a decentralized controller $(G_N(s))$ and R(s) is guaranteed if and only if the following conditions are satisfied [SKOGESTAD, 1996].

Theorem 5

The feedback system in Fig.1 is stable if and only if

- $\det(F(s)) \neq 0 \quad \forall s \in D$
- $N[0, \det(F(s))] = n_l$

where $F(s) = I + G_N(s)R(s)$ is the return difference matrix, $N[0, \det(F(s))]$ denotes the number of anticlockwise encirclements of the point [0,0i] by the Nyquist plot of $\det(F(s))$, n_l is the number of open-loop unstable poles, i.e. of $G_N(s)R(s)$, and $D = \{s = j\omega : \omega \in (-\infty,\infty)\}$. Let us factorize $\det(F(s))$ in Theorem 5 as follows

$$det(F(s)) = det(I + G_N(s)R(s)) = det((R(s)^{-1} + G_d(s) + G_m(s))det(R(s))$$

$$det(F(s)) = det(F_o(s))det(R(s))$$
(21)

where $G_N(s) = G_d(s) + G_m(s)$, $G_d(s) = diag\{G_N(s)\}$ and $F_o(s) = R(s)^{-1} + G_d(s) + G_m(s)$. Existence of $R(s)^{-1}$ is implied by the assumption that $det(R(s)) \neq 0$. Using (21), Theorem 5 reads as follows.

Corollary 1

Closed-loop system comprising the nominal model $G_N(s)$ and the controller R(s) is stable if and only if

•
$$\det(F_o(s)) \neq 0 \quad \forall s \in D$$

•
$$N[0, \det(F_o(s))] + N[0, \det(R(s))] = n_l$$
 (22)

If R(s) has no poles in the open R.H.P , $N[0,\det(R(s))]=0$ and the encirclement condition (22) reduces to

$$N[0, \det(F_o(s))] = n_l \tag{23}$$

Consider a diagonal matrix P(s) such that the following two corollaries hold.

Corollary 2

Let $P(s) = R(s)^{-1} + G_d(s)$. The nominal closed-loop system is stable if either of the £rst two conditions and the third condition are met

- $det(P(s) + G_m(s)) \neq 0 \quad \forall s \in D$
- $N[0, det(P(s) + G_m(s))] = n_m$ where n_m denotes the number of unstable poles of matrix $P(s) + G_m(s)$
- either of the matrices $M_k(s)$, k = a, i, o is stable.

For example, for k = a we obtain

$$M_a(s) = -[P(s) + G_m(s)]^{-1} = -\frac{adj[P(s) + G_m(s)]}{det(P(s) + G_m(s))}$$

The matrix $M_a(s)$ is stable if and only if the closed-loop characteristic polynomial

$$p_c(s) = det(P(s) + G_m(s))$$

has all roots in the left half complex plane.

Remark 1

Corollary 2 implies

$$det(P(s) + G_m(s)) = det(P(s))det(I + P(s)^{-1}G_m(s))$$

Because P(s) is a diagonal matrix numerators of all its entries are to be stable. According to the small gain theorem the necessary and suf£cient condition for closed-loop stability (if both transfer function matrices $P(s)^{-1}$ and $G_m(s)$ are stable) reduces to

$$||P(s)^{-1}G_m(s)|| < 1 \Leftrightarrow \sigma_M(G_m(s)) < \sigma_m(P(s))$$
(24)

Inequality (24) has to be ful£lled for all subsystems.

• In the sequel two methods for selecting the diagonal matrix P(s) are presented. For different entries of $P(s) = diag\{P_i(s)\}_{m \times m}$ the following approach can be applied. Due to that matrices P, R, G_d are diagonal the choice of i-th entry $P_i(s)$ of P(s) is following.

$$P_i(s) = R_i(s)^{-1} + G_{di}(s)$$
 $i = 1, 2, ..., m$

or

$$P_i(s) = \frac{P_{ni}(s)}{P_{di}(s)} = \frac{R_{di}G_{ddi} + G_{dni}R_{ni}}{R_{ni}G_{ddi}}$$

Denote

$$P_{ni} = R_{di}G_{ddi} + G_{dni}R_{ni} = R_{ni}\overline{P_{ni}} \quad P_{di} = R_{ni}G_{ddi}$$

From above equation one obtains the characteristic polynomial in the form

$$1 + R_i(s) \frac{G_{dni}(s) - \overline{P_{ni}(s)}}{G_{ddi}(s)} = 1 + R_i(s) G_{di}^m(s)$$
(25)

where the transfer function of i-th modi£ed subsystem is de£ned as follows

$$G_{di}^{m}(s) = \frac{G_{dni} - \overline{P_{ni}}}{G_{ddi}} \tag{26}$$

and diagonal transfer function matrix P(s)

$$P(s) = \{\frac{\overline{P_{ni}(s)}}{G_{ddi}(s)}\}_{m \times m}$$

where $\overline{P_{ni}(s)}$ is stable polynomial with corresponding degree such that the conditions of Corollaries 2 and 3 are met.

Denote the following polynomials as follows (index i is omitted)

$$p_a(s) = R_d G_{dd} + R_n G_{dn} = a_n s^n + \dots + a_0$$

$$p_b(s) = R_n \overline{P_n} = b_m s^m + \dots + b_0$$

and

$$p_c(s) = p_a(s) - p_b(s) = c_k s^k + \dots + c_0 \quad k = m \quad or \quad k = n$$

The following lemma is important for the next development.

Lemma 1

We are given two stable polynomials $p_b(s)$ and $p_c(s)$. The polynomial $p_a(s)$ will be stable if one of the following condition is met:

• $\varphi(\omega) = |arg(p_c) - arg(p_b)| < \pi \quad \forall \omega \in \Omega$

where

$$\Omega = \{\omega : \omega \in <0 \quad \infty)\}$$

• If for some £nite number of ω_i i = 1, 2, ..., I

$$\varphi(\omega_i) = |arg(p_c) - arg(p_b)| = \pi \quad p_c(\omega_i) \neq -p_b(\omega_i)$$

and for
$$\omega \neq \omega_i$$
, $\varphi(\omega) < \pi$.

Proof. From Zeros exclusion principle [BHATTACHARYYA, 1995] the polynomial $p_a(s)$ will be on the boundary of stability if and only if for some $\omega_i \in \Omega$ and stable polynomials $p_c(s)$ and $p_b(s)$

$$p_a(\omega_i) = 0 \longrightarrow p_c(\omega_i) = -p_b(\omega_i)$$

Because for $\varphi(\omega) = 0 \ \forall \omega \in \Omega$ the polynomial $p_a(s)$ is stable, $p_a(s)$ will be stable if

$$\varphi(\omega) = |arq(p_c) - arq(p_b)| < \pi \quad \forall \omega \in \Omega$$

If for some £nite number of ω_i i = 1, 2, ..., I

$$\varphi(\omega_i) = |arg(p_c) - arg(p_b)| = \pi \quad p_c(\omega_i) \neq -p_b(\omega_i)$$

and for $\omega \neq \omega_i$, $\varphi(\omega) < \pi$ the polynomial $p_a(s)$ is stable.

Remark

From the Lemma 1 and the Mikhailov test stability, see for example [ACKERMAN, 1997] results that for ensure ful£lment of the stability conditions of Lemma 1 the following is recommended:

•
$$|degree(p_c(s)) - degree(p_b(s))| \le 2$$

- stable roots of $p_c(s)$ should be close to stable parts of roots of G_{dd} .
- the controller transfer function numerator has to be a stable polynomial.

Corollary 3

The closed-loop system in Fig.3 is robustly stable if for either of the uncertainty types (2),(3) or (4) satisfying (19) and conditions of Corollary 2 with the corresponding below-given inequalities are met:

• for the additive uncertainty

$$\sigma_M([P(s) + G_m(s)]^{-1}) < \frac{1}{|l_a(s)|}$$

• for the input multiplicative uncertainty

$$\sigma_M([P(s) + G_m(s)]^{-1}G_N(s)) < \frac{1}{|l_i(s)|}$$
(27)

• for the output multiplicative uncertainty

$$\sigma_M(G_N(s)[P(s) + G_m(s)]^{-1}) < \frac{1}{|l_o(s)|}$$

For identical entries of P(s) = p(s)I the following approach has been developed. From Corollary 2 results

$$I + R(s)[G_d(s) - P(s)] = 0 (28)$$

which on the subsystem level represents m characteristic polynomials for individual equivalent subsystems $G_i^{eq}(s)$

$$G_i^{eq} = G_i(s) - p_i(s) \quad i = 1, 2, ..., m$$
 (29)

and controllers $R_i(s)$.

Recall that the characteristic function of $G_m(s)$ are defined as follows

$$det(q_i(s)I - G_m(s)) = 0$$
 $i = 1, 2, ..., m$

If we consider identical entries in the diagonal matrix P(s) = p(s)I, and substitute into the £rst expression of Corollary 2 and equate it to zero

$$det(p(s)I + G_m(s)) = 0 (30)$$

we actually obtain a relation for calculating p(s) as a characteristic function of $[-G_m(s)]$ If for a £xed $l \in \{1, 2, ..., m\}$ p(s) is chosen as $p(s) = -g_l(s)$ then

$$det(F_o(s)) = \prod_{i=1}^{m} [p(s) + g_i(s)] = 0$$
(31)

In that case according to respect to Corollary 2 the closed-loop system has located poles in the left half plane and some on the imaginary axis. Stability conditions of complex system for the case of identical entries of P(s) are given in the following Theorem [Kozáková, 2003].

Theorem 6

The closed-loop system in Fig.1 comprising the system $G_N(s)$ and a not unstable decentralized controller R(s) is stable with a degree of stability $\alpha>0$ if and only if for a selected characteristic function of $-G_m(s-\alpha)$, $p(s-\alpha)=-g_l(s-\alpha)$ there exists a constant α_m such that for all α and any α_1

$$0 \le \alpha_1 < \alpha \le \alpha_m$$

and $\forall s \in D$ the following conditions hold

•
$$det(F_o(s)) = \prod_{i=1}^m [p(s-\alpha) + g_i(s-\alpha_1)] \neq 0$$

$$\sum_{i=1}^{m} N[0, m_{il}^{eq}(s)] = n_m$$
 (32)

where

$$m_{il}^{eq} = [p(s - \alpha) + g_i(s - \alpha_1)]$$
 $i = 1, 2, ..., m$

However, if $\alpha_m \to 0$ and for some $s \in D$ happens that

$$det(F_o(s)) = \prod_{i=1}^{m} [p(s - \alpha) + g_i(s - \alpha_1)] = 0$$

i.e. if the plot of $p(s-\alpha)$ and any characteristic locus $g_i(s-\alpha_1), i=1,2,...,m$ happen to cross, conditions of Theorem 6 are not met and the closed-loop stability cannot be achieved using the decentralized controller R(s). The above partial results are summarized in the following de£nition and theorem.

De£nition 1

For $l \in \{1, 2, ..., m\}$ and $\alpha > \alpha_1 \ge 0$ the characteristic function $g_l(s - \alpha)$ of $[-G_m(s - \alpha)]$ is called a stable characteristic function if it satisfies Theorem 6. The set of all stable characteristic functions is denoted P_S .

Theorem 7

The closed-loop system in Fig.1 comprising the system $G_N(s)$ and a not unstable decentralized controller R(s) is stable with a degree of stability $\alpha > 0$ if and only if:

- $p(s-\alpha) = -g_l(s-\alpha) \in P_S$ $\forall s \in D$ for some £xed $l \in \{1, 2, ..., m\}$ and $\alpha > \alpha_1 \ge 0$
- all equivalent characteristic polynomials (28) are stable with the roots satisfying

$$Res \le -\alpha$$

4 EXAMPLES

In the £rst example the Magnetic levitation model has been considered. The problem is to design a robust PID controller which will guarantee stability and a desired performance in terms of phase margin over the whole operation range of the plant. The magnetic levitation model is described in [HUMUSOFT, 1996-2002] and the linearized model is given as follows

$$\dot{x} = Ax + Bu \quad y = Cx$$

where $x^T = \begin{bmatrix} \Delta x & \Delta x_1 \end{bmatrix}$ and

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{k_{DA}^2 k_f U_{MUD}^2}{m_k (x_{oo} - x_o)^3} & -\frac{k_{fv}}{m_k} \end{bmatrix} \quad B = \begin{bmatrix} 0 \\ -\frac{2k_{DA}^2 k_f U_{MUD}}{m_k (x_{oo} - x_o)^2} \end{bmatrix}$$

$$C = \begin{bmatrix} k_{AD}k_x & 0 \end{bmatrix}$$

The corresponding transfer function

$$y(s) = C(sI - A)^{-1}Bu(s) \to G(s) = \frac{y(s)}{u(s)} = \frac{k_m}{as^2 + bs - 1}$$

For more detail see [HUMUSOFT, 1996-2002].

The linear interval model of the magnetic levitation is given as follows

$$k_m \in <2.4 \quad 6.8> \quad a \in <1.34 \quad 4.025>*10^{-4} \quad b \in <1.7975 \quad 5.3895>*10^{-6}$$

Let the required of closed-loop performance be given in terms of $M_T=1.6$, $M_S=2$ and a phase margin more than $PM=72\ degrees$. Using the extremal transfer function (9)and the Bode approach the robust PID controller transfer function has been obtained

$$R(s) = \frac{.02748s^2 + 1.278s + 8.162}{s}$$

The Bode diagram for the worst case open-loop system is in Fig.4

Nyquist plot with the circle de£ning the prohibited area are in Fig.5

The worst case closed loop step response is given in Fig.6

Applying Corollary 2 and Remark 1 in the robust PID controller design for the above example have obtained the following results. The additive uncertainty $|l_a(s)|$ versus omega plot is depicted in the Fig.7

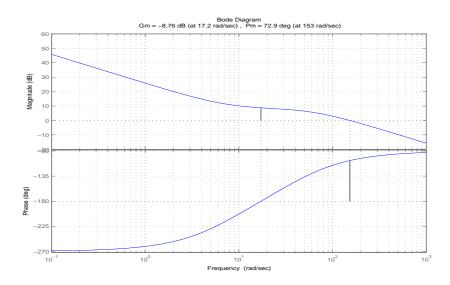


Figure 4 – Bode diagram of open-loop system

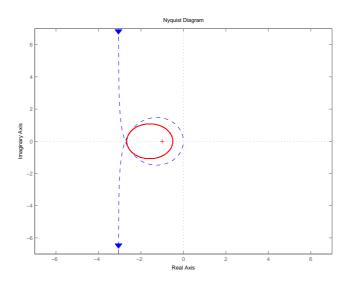


Figure 5 – Nyquist plot with the prohibited area

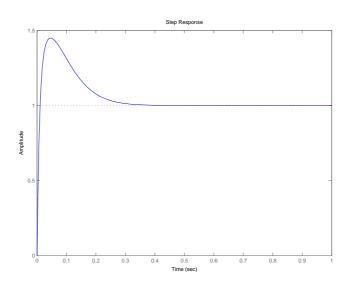


Figure 6 – Closed-loop step response

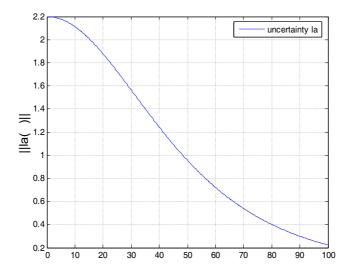


Figure 7 - The additive uncertainty $|l_a(s)|$ versus ω plot

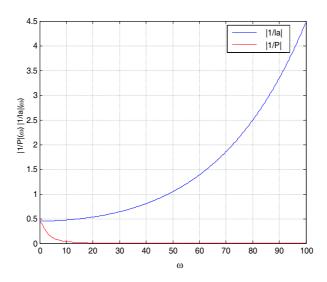


Figure 8 – Veri£cation of the robust stability condition

Taking $\overline{P_n(s)}=(s+60)(s+5)(s+2)$ and applying the D-partition approach the following PID controller is obtained

$$k_p + k_i/s + k_d s = 1.5 + 3/s + 0.03s$$

Veri£cation of the robust stability condition (27) is in the Fig.8; the D-curve for choosing the controller gain k_d is the Fig.9.

Closed-loop step responses in the two plant working points are in Fig.10 and Fig.11 1st working point transfer function, Fig.10:

$$G_1(s) = \frac{6.8}{0.0004025s^2 + 5.389^{-6}s - 1}$$

second working point transfer function, Fig.11:

$$G_2(s) = \frac{2.4}{0.000134s^2 + 1.797^{-6}s - 1}$$

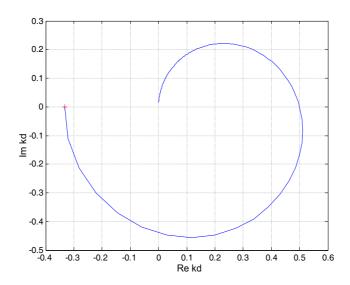


Figure 9 – D-plot for the choice of k_d

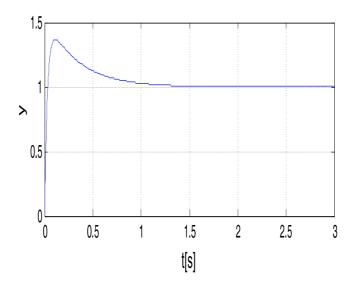


Figure 10 - Closed-loop step response in the 1st working point

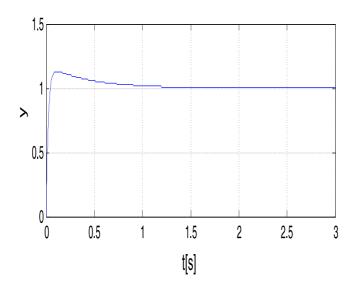


Figure 11 – Closed-loop step response in the 2nd working point

The second example deals with the glass tube drawing plant where the glass metal ¤owing out from feeder is wrapping around a rotating cylindrical blowpipe. At its lower end, a tube is continuously being drawn using a drawing machine situated at the end of the line. Forming air is blown into the tube under a certain pressure. The produced glass tube has to have required parameters: outer diameter and wall thickness; these quantities are manipulated through the pressure of the forming air and the drawing speed of the drawing machine. Assume pairing of the input and output variables de£ning individual subsystems to be completed as follows:

 u_1 - blowing air pressure

 u_2 -speed of drawing

 y_1 -outside diameter of the tube and

 y_2 -tube wall thickness.

The process was linearized in several operating points. The below transfer function matrix corresponds to one chosen operating point.

$$G(s) = \begin{bmatrix} \frac{187e^{-.5s}}{s^2 + 10.6s + 17.2} & \frac{5.45(s - 4.5)}{s^2 + 11.85s + 27.95} \\ \frac{25}{s^2 + 8.84s + 19.52} & \frac{57.5}{s^2 + 13.42s + 39.76} \end{bmatrix}$$

The objective is to design two local decentralized PID controllers guaranteeing that the pre-set output parameters (wall thickness, outside tube diameter) are maintained and the whole process is robustly stable within 15 percent of plant parameter changes. The design procedure is as follows: The characteristic loci (CL) of $G_m(s-\alpha)$ for $\alpha=\{0,.4\}$ are plotted in Fig.12 and Fig.13. One of them has been chosen to generate P(s). Consider the £rst characteristic locus $g1(s-\alpha)$ and specify g(s) to be g(s)=-g1(s-0.4); the corresponding equivalent characteristic loci $m_{i1}^{eq}=[g1(s-.4)-g_i(s),i=1,2]$ are plotted in Fig.14. According to $De\mbox{\it Enition 1}\ g1(s-.4)$ is a stable characteristic locus. Next, the D-partition method has been applied to both equivalent subsystems obtained by modifying the Nyquist

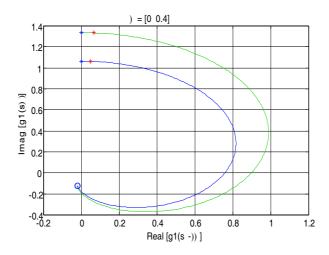


Figure 12 – Characteristic locus $g1(s-\alpha)$ of $G_m(s-\alpha), \alpha=\{0,0.4\}$

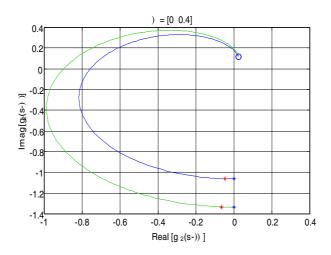


Figure 13 – Characteristic locus $g2(s-\alpha)$ of $G_m(s-\alpha), \alpha=\{0,0.4\}$

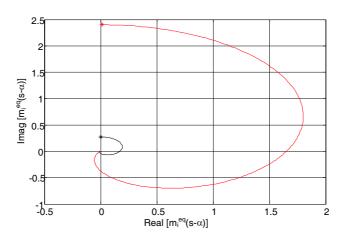


Figure 14 – Equivalent characteristic loci $m_{i1}^{eq}, i=1,2$

plots of decoupled subsystems through the chosen characteristic locus g1(s-.4). Corresponding D-plots in the $(k_p=r0,k_i=r1)$ plane for the £rst and second subsystems are in Fig.15 and Fig 16, respectively

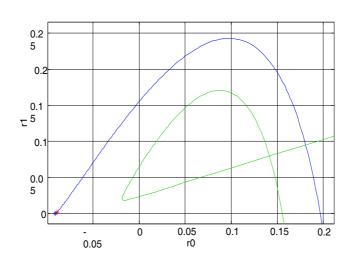


Figure 15 – D-partition of the (r0, r1) plane 1st subsystem

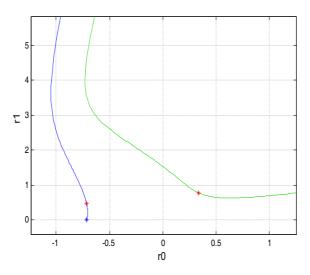


Figure 16 – D-partition of the (r0, r1) plane 2nd subsystem

From the boundary plots of the stable controller parameter regions with degree of stability $\alpha=.4$ the following PI controller parameters have been chosen

$$R_1(s) = .047 + \frac{.0564}{s}$$
 $R_2(s) = 0.3999 + \frac{.7041}{s}$

The closed-loop poles are as follows

$$eigCL = \{-.4232 \pm .1632i; -.8181; -1, 277; -1.3803; \\ -6.4557 \pm 3.55i; -7.3585; -10.7034 \pm 5.1116i; -14.4123\}$$

The above designed local PI controllers guarantee stability of the full nominal closed-loop system with the achieved degree of stability $\alpha=0.4232$. Assume that all parameters of the plant transfer function vary within ± 15 percent around their nominal value;thus the uncertain system can be described by 3 transfer function matrices corresponding to the nominal model, the +15 percent model and the -15 percent model. After evaluating the plant uncertainty using (2), (3), (4) the three plots in Fig.17 have been obtained. To verify robust closed-loop stability under the decentralized controller designed for the nominal model (20) has been modified to give

$$l_k(s)\sigma_M(M_k(s)) < 1 \quad k = a, i, o \tag{33}$$

Fig.18 shows the result of the robust stability test: as all plots (either of them one would suf£ce) lie below 1, the closed-loop system is robustly stable for the 15 percent changes in all plant parameters.

5 CONCLUSION

In this paper a novel design technique is proposed to guarantee a required performance of the full MIMO system by applying the independent design to the equivalent subsystems.

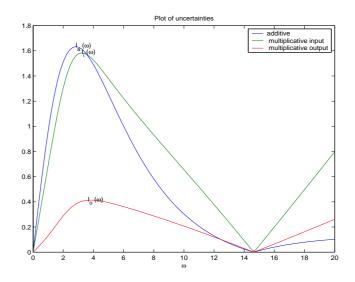


Figure 17 – Plot of three type uncertainties

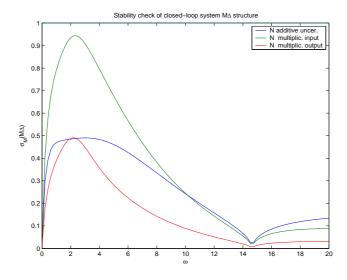


Figure 18 – The robust stability check of closed-loop system

6 ACKNOWLEDGMENT

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